1 Introduction to particle dynamics

In the study of dynamics at an advanced level, it is important to consider many approaches and points of view in order that one may attain a broad theoretical perspective of the subject. As we proceed we shall emphasize those methods which are particularly effective in the analysis of relatively difficult problems in dynamics. At this point, however, it is well to review some of the basic principles in the dynamical analysis of systems of particles. In the process, the kinematics of particle motion will be reviewed, and many of the notational conventions will be established.

1.1 Particle motion

The laws of motion for a particle

Let us consider Newton's three laws of motion which were published in 1687 in his *Principia*. They can be stated as follows:

- I. Every body continues in its state of rest, or of uniform motion in a straight line, unless compelled to change that state by forces acting upon it.
- II. The time rate of change of linear momentum of a body is proportional to the force acting upon it and occurs in the direction in which the force acts.
- III. To every action there is an equal and opposite reaction; that is, the mutual forces of two bodies acting upon each other are equal in magnitude and opposite in direction.

In the dynamical analysis of a system of particles using Newton's laws, we can interpret the word "body" to mean a particle, that is, a certain fixed mass concentrated at a point. The first two of Newton's laws, as applied to a particle, can be summarized by the *law of motion*:

 $\mathbf{F} = m\mathbf{a}$

(1.1)

Here **F** is the total force applied to the particle of mass *m* and it includes both direct contact forces and field forces such as gravity or electromagnetic forces. The acceleration **a** of the particle must be measured relative to an *inertial* or *Newtonian* frame of reference. An example of an inertial frame is an xyz set of axes which is not rotating relative to the "fixed"

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stars and has its origin at the center of mass of the solar system. Any other reference frame which is not rotating but is translating at a constant rate relative to an inertial frame is itself an inertial frame. Thus, there are infinitely many inertial frames, all with constant translational velocities relative to the others. Because the relative velocities are constant, the acceleration of a given particle is the same relative to any inertial frame. The force \mathbf{F} and mass *m* are also the same in all inertial frames, so Newton's law of motion is identical relative to all inertial frames.

Newton's third law, the law of action and reaction, has a corollary assumption that the interaction forces between any two particles are directed along the straight line connecting the particles. Thus we have the *law of action and reaction*:

When two particles exert forces on each other, these interaction forces are equal in magnitude, opposite in sense, and are directed along the straight line joining the particles.

The *collinearity* of the interaction forces applies to all mechanical and gravitational forces. It does not apply, however, to interactions between moving electrically charged particles for which the interaction forces are equal and opposite but not necessarily collinear. Systems of this sort will not be studied here.

An alternative form of the equation of motion of a particle is

$$\mathbf{F} = \dot{\mathbf{p}} \tag{1.2}$$

where the linear momentum of the particle is

$$\mathbf{p} = m\mathbf{v} \tag{1.3}$$

and \mathbf{v} is the particle velocity relative to an inertial frame.

Kinematics of particle motion

The application of Newton's laws of motion to a particle requires that an expression can be found for the acceleration of the particle relative to an inertial frame. For example, the position vector of a particle relative to a *fixed* Cartesian frame might be expressed as

$$\mathbf{r} = x\mathbf{i} + y\mathbf{j} + z\mathbf{k} \tag{1.4}$$

where **i**, **j**, **k** are *unit vectors*, that is, vectors of unit magnitude which have the directions of the positive x, y, and z axes, respectively. When unit vectors are used to specify a vector in 3-space, the three unit vectors are always linearly independent and are nearly always mutually perpendicular. The velocity of the given particle is

$$\mathbf{v} = \dot{\mathbf{r}} = \dot{x}\mathbf{i} + \dot{y}\mathbf{j} + \dot{z}\mathbf{k} \tag{1.5}$$

and its acceleration is

$$\mathbf{a} = \dot{\mathbf{v}} = \ddot{x}\mathbf{i} + \ddot{y}\mathbf{j} + \ddot{z}\mathbf{k} \tag{1.6}$$

relative to the inertial frame.

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A force **F** applied to the particle may be described in a similar manner.

$$\mathbf{F} = F_x \mathbf{i} + F_y \mathbf{j} + F_z \mathbf{k} \tag{1.7}$$

where (F_x, F_y, F_z) are the *scalar components* of **F**. In general, the force components can be functions of position, velocity, and time, but often they are much simpler.

If one writes Newton's law of motion, (1.1), in terms of the Cartesian unit vectors, and then equates the scalar coefficients of each unit vector on the two sides of the equation, one obtains

$$F_x = m\ddot{x}$$

$$F_y = m\ddot{y}$$

$$F_z = m\ddot{z}$$
(1.8)

These three scalar equations are equivalent to the single vector equation. In general, the scalar equations are coupled through the expressions for the force components. Furthermore, the differential equations are often nonlinear and are not susceptible to a complete analytic solution. In this case, one can turn to numerical integration on a digital computer to obtain the complete solution. On the other hand, one can often use energy or momentum methods to obtain important characteristics of the motion without having the complete solution.

The calculation of a particle acceleration relative to an inertial Cartesian frame is straightforward because the unit vectors $(\mathbf{i}, \mathbf{j}, \mathbf{k})$ are fixed in direction. It turns out, however, that because of system geometry it is sometimes more convenient to use unit vectors that are not fixed. For example, the position, velocity, and acceleration of a particle moving along a circular path are conveniently expressed using radial and tangential unit vectors which change direction with position.

As a more general example, suppose that an arbitrary vector **A** is given by

$$\mathbf{A} = A_1 \mathbf{e}_1 + A_2 \mathbf{e}_2 + A_3 \mathbf{e}_3 \tag{1.9}$$

where the unit vectors \mathbf{e}_1 , \mathbf{e}_2 , and \mathbf{e}_3 form a mutually orthogonal set such that $\mathbf{e}_3 = \mathbf{e}_1 \times \mathbf{e}_2$. This unit vector triad changes its orientation with time. It rotates as a rigid body with an angular velocity $\boldsymbol{\omega}$, where the direction of $\boldsymbol{\omega}$ is along the axis of rotation and the positive sense of $\boldsymbol{\omega}$ is in accordance with the right-hand rule.

The first time derivative of A is

$$\dot{\mathbf{A}} = \dot{A}_1 \mathbf{e}_1 + \dot{A}_2 \mathbf{e}_2 + \dot{A}_3 \mathbf{e}_3 + A_1 \dot{\mathbf{e}}_1 + A_2 \dot{\mathbf{e}}_2 + A_3 \dot{\mathbf{e}}_3$$
(1.10)

where

$$\dot{\mathbf{e}}_i = \boldsymbol{\omega} \times \mathbf{e}_i \qquad (i = 1, 2, 3) \tag{1.11}$$

Thus we obtain the important equation

$$\dot{\mathbf{A}} = (\dot{\mathbf{A}})_r + \boldsymbol{\omega} \times \mathbf{A} \tag{1.12}$$

Here $\dot{\mathbf{A}}$ is the time rate of change of \mathbf{A} , as measured in a nonrotating frame that is usually considered to also be inertial. $(\dot{\mathbf{A}})_r$ is the derivative of \mathbf{A} , as measured in a rotating frame in

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which the unit vectors are fixed. It is represented by the first three terms on the right-hand side of (1.10). The term $\boldsymbol{\omega} \times \mathbf{A}$ is represented by the final three terms of (1.10). In detail, if the angular velocity of the rotating frame is

$$\boldsymbol{\omega} = \omega_1 \mathbf{e}_1 + \omega_2 \mathbf{e}_2 + \omega_3 \mathbf{e}_3 \tag{1.13}$$

then

$$\dot{\mathbf{A}} = (\dot{A}_1 + \omega_2 A_3 - \omega_3 A_2) \mathbf{e}_1 + (\dot{A}_2 + \omega_3 A_1 - \omega_1 A_3) \mathbf{e}_2 + (\dot{A}_3 + \omega_1 A_2 - \omega_2 A_1) \mathbf{e}_3$$
(1.14)

Velocity and acceleration expressions for common coordinate systems

Let us apply the general equation (1.12) to some common coordinate systems associated with particle motion.

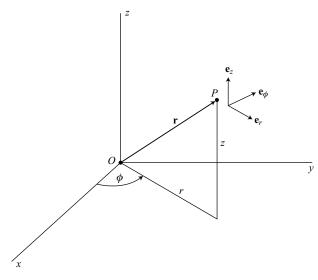
Cylindrical coordinates

Suppose that the position of a particle *P* is specified by the values of its cylindrical coordinates (r, ϕ, z) . We see from Fig. 1.1 that the position vector **r** is

$$\mathbf{r} = r\mathbf{e}_r + z\mathbf{e}_z \tag{1.15}$$

where we notice that r is not the magnitude of **r**. The angular velocity of the $\mathbf{e}_r \mathbf{e}_{\phi} \mathbf{e}_z$ triad is

$$\boldsymbol{\omega} = \dot{\boldsymbol{\phi}} \mathbf{e}_z \tag{1.16}$$





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so we find that $\dot{\mathbf{e}}_z$ vanishes and

$$\dot{\mathbf{e}}_r = \boldsymbol{\omega} \times \mathbf{e}_r = \dot{\phi} \mathbf{e}_\phi \tag{1.17}$$

Thus, the velocity of the particle P is

$$\mathbf{v} = \dot{\mathbf{r}} = \dot{r}\mathbf{e}_r + r\dot{\phi}\mathbf{e}_\phi + \dot{z}\mathbf{e}_z \tag{1.18}$$

Similarly, noting that

$$\dot{\mathbf{e}}_{\phi} = \boldsymbol{\omega} \times \mathbf{e}_{\phi} = -\dot{\phi}\mathbf{e}_r \tag{1.19}$$

we find that its acceleration is

$$\mathbf{a} = \dot{\mathbf{v}} = (\ddot{r} - r\dot{\phi}^2)\,\mathbf{e}_r + (r\ddot{\phi} + 2\dot{r}\dot{\phi})\,\mathbf{e}_\phi + \ddot{z}\mathbf{e}_z \tag{1.20}$$

If we restrict the motion such that \dot{z} and \ddot{z} are continuously equal to zero, we obtain the velocity and acceleration equations for plane motion using *polar coordinates*.

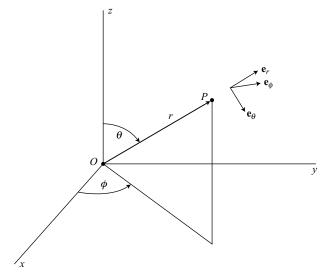
Spherical coordinates

From Fig. 1.2 we see that the position of particle *P* is given by the spherical coordinates (r, θ, ϕ) . The position vector of the particle is simply

$$\mathbf{r} = r\mathbf{e}_r \tag{1.21}$$

The angular velocity of the $\mathbf{e}_r \mathbf{e}_{\theta} \mathbf{e}_{\phi}$ triad is due to $\dot{\theta}$ and $\dot{\phi}$ and is equal to

$$\boldsymbol{\omega} = \dot{\phi}\cos\theta \,\mathbf{e}_r - \dot{\phi}\sin\theta \,\mathbf{e}_\theta + \dot{\theta} \,\mathbf{e}_\theta \tag{1.22}$$





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We find that

$$\dot{\mathbf{e}}_{r} = \boldsymbol{\omega} \times \mathbf{e}_{r} = \dot{\theta} \mathbf{e}_{\theta} + \dot{\phi} \sin \theta \, \mathbf{e}_{\phi}$$

$$\dot{\mathbf{e}}_{\theta} = \boldsymbol{\omega} \times \mathbf{e}_{\theta} = -\dot{\theta} \mathbf{e}_{r} + \dot{\phi} \cos \theta \, \mathbf{e}_{\phi}$$

$$\dot{\mathbf{e}}_{\phi} = \boldsymbol{\omega} \times \mathbf{e}_{\phi} = -\dot{\phi} \sin \theta \, \mathbf{e}_{r} - \dot{\phi} \cos \theta \, \mathbf{e}_{\theta}$$

(1.23)

Then, upon differentiation of (1.21), we obtain the velocity

$$\mathbf{v} = \dot{\mathbf{r}} = \dot{r}\mathbf{e}_r + r\dot{\theta}\mathbf{e}_\theta + r\dot{\phi}\sin\theta \,\mathbf{e}_\phi \tag{1.24}$$

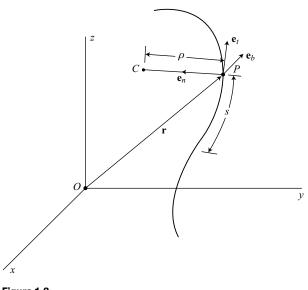
A further differentiation yields the acceleration

$$\mathbf{a} = \dot{\mathbf{v}} = (\ddot{r} - r\dot{\theta}^2 - r\dot{\phi}^2 \sin^2 \theta) \mathbf{e}_r + (r\ddot{\theta} + 2\dot{r}\dot{\theta} - r\dot{\phi}^2 \sin \theta \cos \theta) \mathbf{e}_\theta + (r\ddot{\phi}\sin\theta + 2\dot{r}\dot{\phi}\sin\theta + 2r\dot{\theta}\dot{\phi}\cos\theta) \mathbf{e}_\phi$$
(1.25)

Tangential and normal components

Suppose a particle *P* moves along a given path in three-dimensional space. The position of the particle is specified by the single coordinate *s*, measured from some reference point along the path, as shown in Fig. 1.3. It is convenient to use the three unit vectors $(\mathbf{e}_t, \mathbf{e}_n, \mathbf{e}_b)$ where \mathbf{e}_t is tangent to the path at *P*, \mathbf{e}_n is normal to the path and points in the direction of the center of curvature *C*, and the binormal unit vector is

$$\mathbf{e}_b = \mathbf{e}_t \times \mathbf{e}_n \tag{1.26}$$





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The velocity of the particle is equal to its speed along its path, so

$$\mathbf{v} = \dot{\mathbf{r}} = \dot{s}\mathbf{e}_t \tag{1.27}$$

If we consider motion along an infinitesimal arc of radius ρ surrounding P, we see that

$$\dot{\mathbf{e}}_t = \frac{\dot{s}}{\rho} \mathbf{e}_n \tag{1.28}$$

Thus, we find that the acceleration of the particle is

$$\mathbf{a} = \dot{\mathbf{v}} = \ddot{s}\mathbf{e}_t + \dot{s}\dot{\mathbf{e}}_t = \ddot{s}\mathbf{e}_t + \frac{\dot{s}^2}{\rho}\mathbf{e}_n \tag{1.29}$$

where ρ is the radius of curvature. Here \ddot{s} is the tangential acceleration and \dot{s}^2/ρ is the centripetal acceleration. The angular velocity of the unit vector triad is directly proportional to \dot{s} . It is

$$\omega = \omega_t \mathbf{e}_t + \omega_b \mathbf{e}_b \tag{1.30}$$

where ω_t and ω_b are obtained from

$$\dot{\mathbf{e}}_{t} = \omega_{b} \mathbf{e}_{n} = \frac{\dot{s}}{\rho} \mathbf{e}_{n}$$

$$\dot{\mathbf{e}}_{b} = -\omega_{t} \mathbf{e}_{n} = \dot{s} \frac{d\mathbf{e}_{b}}{ds}$$
(1.31)

Note that $\omega_n = 0$ and also that $d\mathbf{e}_b/ds$ represents the *torsion* of the curve.

Relative motion and rotating frames

When one uses Newton's laws to describe the motion of a particle, the acceleration **a** must be *absolute*, that is, it must be measured relative to an inertial frame. This acceleration, of course, is the same when measured with respect to any inertial frame. Sometimes the motion of a particle is known relative to a rotating and accelerating frame, and it is desired to find its absolute velocity and acceleration. In general, these calculations can be somewhat complicated, but for the special case in which the moving frame *A* is not rotating, the results are simple. The absolute velocity of a particle *P* is

$$\mathbf{v}_P = \mathbf{v}_A + \mathbf{v}_{P/A} \tag{1.32}$$

where \mathbf{v}_A is the absolute velocity of any point on frame *A* and $\mathbf{v}_{P/A}$ is the velocity of particle *P* relative to frame *A*, that is, the velocity recorded by cameras or other instruments fixed in frame *A* and moving with it. Similarly, the absolute acceleration of *P* is

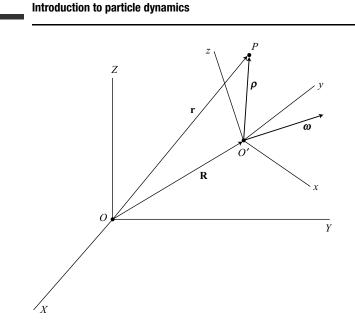
$$\mathbf{a}_P = \mathbf{a}_A + \mathbf{a}_{P/A} \tag{1.33}$$

where we note again that the frame A is moving in pure translation.

Now consider the general case in which the moving xyz frame (Fig. 1.4) is translating and rotating arbitrarily. We wish to find the velocity and acceleration of a particle *P* relative

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to the inertial *XYZ* frame in terms of its motion with respect to the noninertial *xyz* frame. Let the origin O' of the *xyz* frame have a position vector **R** relative to the origin O of the *XYZ* frame. The position of the particle P relative to O' is ρ , so the position of P relative to *XYZ* is

$$\mathbf{r} = \mathbf{R} + \boldsymbol{\rho} \tag{1.34}$$

The corresponding velocity is

$$\mathbf{v} = \dot{\mathbf{r}} = \dot{\mathbf{R}} + \dot{\rho} \tag{1.35}$$

Now let us use the basic equation (1.12) to express $\dot{\rho}$ in terms of the motion relative to the moving *xyz* frame. We obtain

$$\dot{\rho} = (\dot{\rho})_r + \omega \times \rho \tag{1.36}$$

where ω is the angular velocity of the *xyz* frame and $(\dot{\rho})_r$ is the velocity of *P* relative to that frame. In detail,

$$\boldsymbol{\rho} = x\mathbf{i} + y\mathbf{j} + z\mathbf{k} \tag{1.37}$$

and

$$(\dot{\boldsymbol{\rho}})_r = \dot{\boldsymbol{x}}\mathbf{i} + \dot{\boldsymbol{y}}\mathbf{j} + \dot{\boldsymbol{z}}\mathbf{k} \tag{1.38}$$

where \mathbf{i} , \mathbf{j} , \mathbf{k} are unit vectors fixed in the *xyz* frame and rotating with it. From (1.35) and (1.36), the absolute velocity of *P* is

$$\mathbf{v} = \dot{\mathbf{r}} = \mathbf{R} + (\dot{\rho})_r + \omega \times \rho \tag{1.39}$$

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The expression for the inertial acceleration **a** of the particle is found by first noting that

$$\frac{a}{dt}(\dot{\rho})_r = (\ddot{\rho})_r + \omega \times (\dot{\rho})_r \tag{1.40}$$

$$\frac{d}{dt}(\boldsymbol{\omega} \times \boldsymbol{\rho}) = \dot{\boldsymbol{\omega}} \times \boldsymbol{\rho} + \boldsymbol{\omega} \times ((\dot{\boldsymbol{\rho}})_r + \boldsymbol{\omega} \times \boldsymbol{\rho})$$
(1.41)

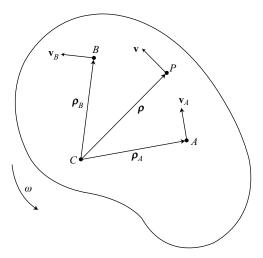
Thus, we obtain the important result:

$$\mathbf{a} = \dot{\mathbf{v}} = \ddot{\mathbf{R}} + \dot{\boldsymbol{\omega}} \times \boldsymbol{\rho} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \boldsymbol{\rho}) + (\ddot{\boldsymbol{\rho}})_r + 2\boldsymbol{\omega} \times (\dot{\boldsymbol{\rho}})_r$$
(1.42)

where ω is the angular velocity of the *xyz* frame. The nature of the various terms is as follows. **R** is the inertial acceleration of O', the origin of the moving frame. The term $\dot{\omega} \times \rho$ might be considered as a tangential acceleration although, more accurately, it represents a changing tangential velocity $\omega \times \rho$ due to changing ω . The term $\omega \times (\omega \times \rho)$ is a centripetal acceleration directed toward an axis of rotation through O'. These first three terms represent the acceleration of a point coincident with P but fixed in the *xyz* frame. The final two terms add the effects of motion relative to the moving frame. The term $(\ddot{\rho})_r$ is the acceleration of P relative to the *xyz* frame, that is, the acceleration of the particle, as recorded by instruments fixed in the *xyz* frame and rotating with it. The final term $2\omega \times (\dot{\rho})_r$ is the *Coriolis acceleration* due to a velocity relative to the moving frame. Equation (1.42) is particularly useful if the motion of the particle relative to the moving *xyz* frame is simple; for example, linear motion or motion along a circular path.

Instantaneous center of rotation

If each point of a rigid body moves in planar motion, it is useful to consider a *lamina*, or slice, of the body which moves in its own plane (Fig. 1.5). If the lamina does not move in pure translation, that is, if $\omega \neq 0$, then a point C exists in the lamina, or in an imaginary





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extension thereof, at which the velocity is momentarily zero. This is the *instantaneous center of rotation*.

Suppose that arbitrary points *A* and *B* have velocities \mathbf{v}_A and \mathbf{v}_B . The instantaneous center *C* is located at the intersection of the perpendicular lines to \mathbf{v}_A and \mathbf{v}_B . The velocity of a point *P* with a position vector $\boldsymbol{\rho}$ relative to *C* is

 $\mathbf{v} = \boldsymbol{\omega} \times \boldsymbol{\rho} \tag{1.43}$

where ω is the angular velocity vector of the lamina. Thus, if the location of the instantaneous center is known, it is easy to find the velocity of any other point of the lamina at that instant. On the other hand, the acceleration of the instantaneous center is generally not zero. Hence, the calculation of the acceleration of a general point in the lamina is usually not aided by a knowledge of the instantaneous center location.

If there is planar rolling motion of one body on another fixed body without any slipping, the instantaneous center lies at the contact point between the two bodies. As time proceeds, this point moves with respect to both bodies, thereby tracing a path on each body.

Example 1.1 A wheel of radius r rolls in planar motion without slipping on a fixed convex surface of radius R (Fig. 1.6a). We wish to solve for the acceleration of the contact point on the wheel. The contact point C is the instantaneous center, and therefore, the velocity of the wheel's center O' is

$$\mathbf{v} = r\omega \mathbf{e}_{\phi}$$
 (1.44)

